# Modeling and simulation of phase change process in Ice Thermal Energy Storage

Tushar Sharma<sup>1</sup>, Dr. Pankaj Kalita<sup>2</sup>

1. Centre for Energy, IIT Guwahati, Guwahati 781039, Assam, India

2. Centre for Energy, IIT Guwahati, Guwahati 781039, Assam, India

## Abstract

The focus of this paper is to study the effect of phase change process of water as it undergoes heat exchange with the Refrigerant, R-22 as the heat transfer fluid. For the purpose of simulation of phase change process, COMSOL Multiphysics ® 5.3a has been used utilizing the inbuilt geometry builder. The motivation for this study comes from the experiments carried out on a lab-scale model of a small ice thermal storage for a fixed time duration. The boundary conditions used for simulation are similar to the conditions used in the actual experimental study. However, an actual comparison of both the studies is beyond the scope of the present study and needs further study and analysis. The reason for this is that the 2D model developed for the present computational study does not take into account the complexities present in the actual 3D model used for the experimental study. The present study enables us to validate the concept used in the actual experimental setup so that we can develop complex 3D userdefined functions to take into account the various complexities associated in the heat transfer process with much more accuracy.

**Key words**: 2D model, Phase Change, Heat Transfer, Fluid flow, Ice thermal energy storage, COMSOL Multiphysics ®

### Introduction

Ice Thermal Energy Storage is a form of Latent Heat Thermal Energy Storage in which water is used as the Phase Change Material, which undergoes phase transformation during charging and discharging periods of operation.

Present study is focused on the phase change simulation using CFD analysis for the 2D model developed in the COMSOL Multiphysics <sup>®</sup> software. Following figure illustrates the schematic of the 2D model used for modeling and simulation purpose of the fluid flow and heat transfer used in the software.



Figure 1: Schematic representation of the 2D model used for simulation

Following	table	shows	the	values	of	various	parameters
selected for	r dimei	nsioning	; of 2	D mode	1.		

Parameter	Value (mm)
Di	10
Do	32
L	100
t	1

Table 1: Parameter values for dimensioning 2D model

The geometry has been created using the inbuilt geometry builder feature and the fine mesh feature is used to generate the mesh used for the CFD simulation study. Following figure shows the Mesh generated using the mesh generate feature.

Figure 2: Fine mesh generated for the purpose of CFD simulation

Fol	lowing	table s	hows th	ne details	s of the	Mesh	generated.	
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Characteristic	Value
Triangular elements	8272
Quadrilateral elements	1344
Edge elements	754
Vertex elements	12
Average element quality	0.827
Element area ratio	0.066
Mesh area	3200 mm <sup>2</sup>
Average growth rate	1.379

Table 2: Mesh details for the mesh generated for CFD simulation

#### Theory

Latent Heat Thermal Energy storage (LHTES) forms the basic mechanism of operation of Ice Thermal Energy storage system. The way it works is illustrated in the figure 3 below.



Figure 3: The three processes in a general CTES, Charging, Storing and Discharging.

### **Charging process**

In the Charging process, the heat transfer fluid at high temperature enters the chamber where storage fluid or the Phase Change Material is stored. The melting process occurs and the temperature of the PCM starts to rise with time, till the desired temperature is achieved at the outlet of the heat transfer fluid.

### Storing process

In the Storing process, Energy accumulation occurs for a very brief time in which the storage is being left idle. QI represents the heat load infiltrating into the system from surroundings.

### **Discharging process**

In the discharging process, the solidification/ freezing of the PCM occurs. The heat transfer fluid enters the chamber where PCM is stored at a temperature lower than what is to be maintained in the PCM and takes away the heat of fusion. Thus, making the PCM cooler in the process. Heat transfer fluid becomes relatively hotter, however the temperature at the outlet is maintained in the range, which is needed inside the Cold Storage prototype cabinet.

The boundary conditions used for simulation of heat transfer with phase change are listed in the table below.

Parameter	Value
Discharging T <sub>in</sub>	-10 °C
Discharging T <sub>out</sub>	-4 °C
Storage Fluid (Water) Temp (Discharge)	0 °C
Charging T <sub>in</sub>	6 °C
Charging T <sub>out</sub>	-2 °C
Storage Fluid (Water) Tenp (Charging)	-6 °C
Heat transfer fluid (R-22) mass flow rate	0.00159 kg/s

**Table 3:** Input parameters for discharging and charging periods of Ice

 Thermal Storage

# Governing Equations used for Simulation of heat transfer with phase change

Conjugated heat transfer method is used as the physics that best describes our case for simulation, as the heat transfer is taking place at the interface between copper tube carrying the Refrigerant ( heat transfer fluid ) which is flowing and the Storage fluid ( water) which is stationary and changing its phase from one phase to the other in both the processes. Following set of governing equations have been used for simulation of the heat transfer process with phase change.

$$\rho \frac{\partial u}{\partial t} - \nabla . \eta (\nabla u + (\nabla u)^T) + \rho (u. \nabla) u + \nabla p = 0$$
(1)

$$\nabla . u = 0 \tag{2}$$

$$\rho C_p \frac{\partial T}{\partial t} + \rho C_p u. \nabla T + \nabla T + \nabla q = Q + q_0 + Q_p + Q_{\nu d}(3)$$

$$q = -k\nabla T \tag{4}$$

$$C_{p} = \frac{1}{\rho} \left( \theta \rho_{phase1} C_{phase1} + (1 - \theta) \rho_{phase2} C_{p,phase2} \right) + L \frac{\partial \alpha_{m}}{\partial T}$$
(5)

$$k = \theta k_{phase1} + (1 - \theta) k_{phase2} \tag{6}$$

$$\alpha_m = \frac{1}{2} \frac{(1-\theta)\rho_{phase2} - \theta\rho_{phase1}}{\theta\rho_{phase1} + (1-\theta)\rho_{phase2}}$$
(7)

In the above equations,  $\rho$  is the density of the fluid, u is the velocity of the flowing fluid, p is the pressure, Q is the heat flux, T is the temperature,  $C_p$  is the specific heat capacity,  $\theta$  is the melt fraction, k is the equivalent thermal conductivity, L is the latent heat energy,  $\alpha$  is the thermal diffusivity.

Following set of properties were used for the phase change fluid in different processes.

Mat. Properties	Ice	Water
Density (kg/m <sup>3</sup> )	918	997
Specific Heat Capacity ( J/kgK)	2052	4179
Thermal conductivity (W/mK)	2.31	0.613

**Table 4:** Material properties of water and ice used for simulation of phase change.

### Simulation results and discussions

For the purpose of simulation, conjugated heat transfer model is used as the physics. Time dependent study is used with a total runtime of 4100 secs and a time step of 1 sec. The physics is solved for a total of 27345 degree of freedoms. The physics is solved for three variables, namely pressure (P), velocity (V) and temperature (T). Coupled direct solver PARDISO is used as the solver by COMSOL.

### 1. Charging Period

Following results have been observed for the Charging period. Three cut lines have been defined for the evaluation of properties along the length of the conductor. Details of the cut line are shown in the table given below:

	*	
Cut Line 1	X = -50 to $X = 50$	Y = 0
Cut Line 2	X = -50 to $X = 50$	Y = 10
Cut Line 2	Y = 50 to $Y = 50$	V - 10



**Figure 4:** Variation of temperature with time across the length of the 2D model a) Mid-section b) 10 mm above the x-axis c) 10 mm below the x-axis.

Following figure shows the phase transformation during the charging period.



**Figure 5**: Phase transformation with time across the length of the 2D model a) 10 mm above the x-axis b) 10 mm below the x-axis.

It can be seen from the Figure 4 that the Temperature is increasing with time for the two cut sections inside the storage fluid zones in b) and c), while the opposite is seen in the a) inside the heat transfer fluid (R-22) as expected from the heat transfer physics. However, while moving from the left end to the right end, the temperature is reducing at any given time, which is expected, as the temperature to be maintained is lesser in right end to meet the requirements of boundary conditions.

Looking for the phase transformation point of view in the Figure 5 at two cut sections a) and b), it can be said that the phase is being transformed into liquid from solid where 0 is liquid and 1 is solid. With time, transformation is taking place as expected from the heat transfer physics. However, more melting is taking place in the upper section than the lower section, which can be investigated in the future for further study.

### 2. Discharging period

Following results have been observed for the Charging period. Three cut lines have been defined for the evaluation of properties along the length of the conductor. Details of the cut line are shown in the table given below:

	0	
Cut Line 1	X = -50 to $X = 50$	<b>Y</b> = 0
Cut Line 2	X = -50 to $X = 50$	Y = 10
Cut Line 3	X = -50 to $X = 50$	Y = - 10



**Figure 6**: Variation of temperature with time across the length of the 2D model a) Mid-section b) 10 mm above the x-axis c) 10 mm below the x-axis.

Following figure shows the phase transformation during the charging period.



**Figure 7:** Phase transformation with time across the length of the 2D model a) 10 mm above the x-axis b) 10 mm below the x-axis.

It can be seen from the Figure 4 that the Temperature is reducing with time for the two cut sections inside the storage fluid zones in b) and c), while the opposite is seen in the a) inside the heat transfer fluid (R-22) as expected from the heat transfer physics. However, while moving from the left end to the right end, the temperature is increasing at any given time, which is expected, as the temperature to be maintained is higher in right end to meet the requirements of boundary conditions.

Looking for the phase transformation point of view in the Figure 5 at two cut sections a) and b), it can be said that the phase is being transformed into solid from liquid where 0 is liquid and 1 is solid. With time, transformation is taking place as expected from the heat transfer physics. Similar phase transformation characteristics are observed in both the halves of the 2D model.

# Conclusions

From the study conducted for the phase change heat transfer, it can be concluded that the heat transfer physics have been accurately applied for both the cases viz. charging and discharging of the ice thermal energy storage. Further study can be done on more complex 3D models to accurately predict the phase transfer physics. The computational time needed for the simulation was 20 min 20 s for the charging phase and 10 min 48 s on a 4GB RAM computer. For solving more complicated problems, better computational facilities can be used to reduce the computational time and produce faster results.

### References

- Armand Fopah Lele, Thomas Ronnebeck, Christian Rohde, Thomas Schmidt, Frederic Kuznik, Wolfgang K.L.Ruck, Modeling of heat exchangers based on thermochemical material for solar heat storage systems, Energy Procedia 61 (2014) 2809-2813.
- 2. Mandhapati Raju, Sudarshan Kumar, Modeling of a helical coil heat exchanger for Sodium Alanate based On-board Hydrogen Storage system, Excerpt from proceedings of the COMSOL Conference 2010 Boston.
- Andrea Mammoli, Matthew Robinson, Numerical analysis of heat transfer process in a low-cost, high-performance ice storage device for residential applications, Applied Thermal Engineering 128 (2018), 453-463.

### Acknowledgement

We would like to acknowledge the Centre for Energy, IIT Guwahati for lending their computational facility for the purpose of simulation.